CONDITION MONITORING OF VARIABLE SPEED WORM GEARBOX LUBRICATED WITH CONTAMINATED GEAR OIL

MAZNAN BIN ISMON

PERPUSIAKAAN

Faculty of Mechanical and Manufacturing Engineering Universiti Tun Hussein Onn Malaysia

JUNE 2013

To my precious Allah S.W.T, who gave me new life, hope and purpose of life. To Abah and Mak, who giving all they have. To my beloved wife, Sabarina Che Mat

who supported me each step of the way.

and,

To my beloved sons, Danish Danial and Haziq Hazman,

who being my inspiration and motivation in completing this thesis

ACKNOWLEDGEMENT

Above all, I thank Allah S.W.T for his grace and strength given to me so that this research could be made possible. Secondly, I would like to extend my thankfulness to my research supervisor, Prof. Mohd Imran bin Ghazali for his continuous understanding, encouragement and trust in me throughout the entire project. I am grateful for his constructive comments and careful review of all the reports. Any remaining errors are my responsibility.

I also wish to thank all the fellow technician in Noise and Vibrations Laboratory, En. Zainorin bin Kasron, who had given his time and assistance in performing tests.

Finally, my gratitude goes to Universiti Tun Hussein Onn Malaysia and Continuing Education Centre for the kind attention, without this I may have the difficulties to further my study and to complete this project.



ABSTRACT

This research focuses on the condition monitoring of worm gearbox lubricated with contaminated gear oil. The main objectives are to characterize vibration behavior of worm gear as function contaminated lubricant viscosity. Worm gearbox consists of a spirally grooved screw moving against a toothed wheel. Experimental test rig with inverter, electric motor and gearbox to be develop and employed for the investigation. New gear oil with ISO VG 100, 460 and 680 had been indentified to serve the sliding friction of both gears as a comparison to contaminated oil. A predetermine speed of electric motor to be introduced to the gearbox and a set of vibration instrumentations was installed as to capture the vibration signal generated. As to enhance the effectiveness of condition monitoring techniques, temperature rising in the gear oil upon the testing period are monitored using PicoLog Data Logger and later to be analyze. The experiment had revealed that oil with higher viscoscity will contribute to less vibration spectrum. At 900 RPM, the vibration had reduced for about 0.0438 mm/s for ISO VG 100 during the experimental testing within new and contaminated gear oil. It has contribute a reduction of vibration about 51.02% while the temperature monitoring had shown on the higher side compare to the new oil with less viscoscity.



TABLE OF CONTENTS

DECLARATION		
DEDICATION		iii
ACKNOWLED	GEMENT	iv
ABSTRACT		v
CHAPTER 1		1
INTRODUCTIO	DN	1
1.1	Introduction	1
1.2	Problem Statement	2
1.3	Objective	3
1.4	Scope of study	3
1.5	Expected Result	4
1.6	Potential Contribution	5
CHAPTER 2		6
LITERATURE	REVIEW AND THEORY	6
2.1	Introduction	6
2.2	Condition Monitoring Techniques	7
2.2	2.1 Vibration Analysis	9
2.2	2.2 Oil Sampling Analysis	9
2.2	2.3 Thermograph Monitoring	10
2.2	2.4 Acoustic Monitoring	10
2.2	E.5 Electric Motor Current Monitoring	11
2.3	Industrial Gearboxes	11
2.3	.1 Worm gear	13
2.3	9.2 Planetary Gear Transmission System	14

2.3.3	Parallel Gear Transmission System	14	
Lubr	icant	15	
2.4.1	Lubricant Viscosity Classification	15	
2.4.2	The Impact of lubricant on mechanical vibration	17	
2.4.3	Contamination Effect	18	
2.4.4	Identifying lubricant Contamination	18	
2.4.5	Lubricant Life Time	19	
2.4.6	Oxidation in Lubricant	20	
2.4.7	Oil Analysis	21	
2.4.8	Wear particle analysis for machine condition	21	
2.4.9	Lubrication viscosity	22	
Theo	ory	23	
2.5.1	Introduction	23	
2.5.2	Detection Defects with Vibration analysis	23	
2.5.3	Vibration Standard and Criteria	27	
;		30	
CHAPTER 330METHODOLOGY30			
Intro	duction	30	
Prepa	aring Test Rig/Experimental setup	31	
Lubr	icant of Gearbox	33	
Instr	umentations	34	
Proc	edure and equipment	37	
Ļ		38	
		38	
Intro	duction	38	
Rota	tional speed of output and input rotor	39	
Vibr	ation analysis	40	
Tem	perature Analysis	45	
	Lubr 2.4.1 2.4.2 2.4.3 2.4.4 2.4.5 2.4.6 2.4.7 2.4.8 2.4.9 Theo 2.5.1 2.5.2 2.5.3 LOGY Intro Prep Lubr Instr Proc	Lubricant2.4.1Lubricant Viscosity Classification2.4.2The Impact of lubricant on mechanical vibration2.4.3Contamination Effect2.4.4Identifying lubricant Contamination2.4.5Lubricant Life Time2.4.6Oxidation in Lubricant2.4.7Oil Analysis2.4.8Wear particle analysis for machine condition2.4.9Lubrication viscosityTheory2.5.1Introduction2.5.2Detection Defects with Vibration analysis2.5.3Vibration Standard and CriteriaJOGYIntroductionPreparing Test Rig/Experimental setupLubricant of GearboxInstrumentationsProcedure and equipmentProcedure and equipment	

4.5	Viscosity	48
4.6	Worm Gear Surface	49
4.7	Thermograph	51
CHAPTER 5		53
CONCLUSION	S	53
5.1	Introduction	53
5.2	Conclusion	54
REFERENCES		56
APPENDIX A		58
APPENDIX B		60
APPENDIX C		62
APPENDIX D		95 102
APPENDIX E		102

CHAPTER 1

INTRODUCTION

1.1 Introduction

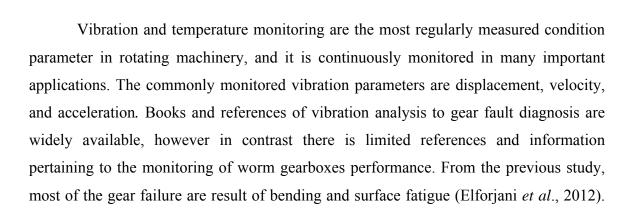


Industrial operations are much depending on the rotating equipment to ensure continuous plant operation or production performance. The internal mechanical parts which have widely contributed to a major breakdown in rotating equipment are known as gears and bearings. On the other hand, lubricants are another factor to be considered as to ensure the performance of gears and bearings are within optimum operating conditions. Any unscheduled breakdown can easily burst the maintenance cost, lapses in production schedule and even interfere the human casualties. As to minimize the unscheduled breakdown, a proper maintenance practice and monitoring systems are required to detect faults in the early stages and provide alert notification to the operators. The capabilities of Condition Monitoring Systems provided it has been conducted correctly could extend the life span of the rotating equipment. These could also reduce the maintenance cost and avoid catastrophic failure to the gears and bearings.

Vibration and temperature monitoring are the techniques for machinery maintenance and fault diagnosis that could predict the gears and bearings failure. Both techniques have their unique advantages and disadvantages associated with the monitoring and fault diagnosis of machinery. When these techniques are conducted independently, only a portion of machine faults are typically diagnosed. However, practical experience has shown that by integrating these two techniques in a machine condition monitoring program will provides better and more reliable information, bringing significant cost benefits to industry.

1.2 Problem Statement

Gearboxes are classified as the rotating equipments at any industrial applications worldwide and play an important role to the industrial performance. The major internal components of the gearboxes are mainly named as shaft, gear, bearing and seal. The challenge is now to seek further significant improvement in the technology and prolong the life span of the internal component. The issue of wear and tear of the gears and bearing are much related to the lubricant performance. Failures associated with gears typically caused by gradual deterioration and wear. Such deterioration probably could happened when rotating equipment are working within a condition of excessive load, excessive vibration, high temperature, lack of lubrication or contaminated lubricant. Such degradation process can be identify if a proper condition monitoring techniques is used to detect and give sign of early warning.





Worm gearbox normally consist of one step speed reduction mechanism but capable for higher reduction ratio. Worm gearbox could be both slow speed and high speed rotating parts internally. Since the vibration analysis could predict the gear failure effectively, the application need to be extend to the output shaft of worm gearboxes which typically has great speed reduction compared to the input shaft. The limitation of vibration analysis to low speed rotating machine has been described in Condition Monitoring of Worm Gears (Elforjani *et al.*, 2012). Further study still need to be conducted to analyse the correlation between vibration, worm gear speed, lubricant characteristic and temperature.

1.3 Objective

The objective of this experiment is to characterize worm gears vibration behavior tested with contaminated lubricant provided new oil as a baseline value. The analysis to be extended up to the lubricant temperature and seek correlations within mechanical vibration.

1.4 Scope of study

Scope of this study is to investigate the correlation between vibration and temperature behavior within different gear oil viscosity characterisation. In order to obtain this, this research will cover the following:

- (i) Proposed the proper experimental rig to be set up related to vibrations issues.
- (ii) Study the viscosity classification of gear oil and preparing samples of contaminated gear oil.

- (iii) To investigate different operating conditions of an experimental rig, consisting of a worm gearbox driven by an inverted electric motor.
- (iv) To perform experimental test to obtain both data.
- (v) To assess the sampling oil analysis characterisation for the contaminated oil.
- (vi) To analyze the data and find the correlations of mechanical vibration and temperature.

1.5 Expected Result

From the experimental rig, the worm gearbox will initially run under normal operating conditions as a comparative test. A series of tests were then conducted corresponding to a different input speed using the same contaminated lube oil. Vibration data will be regularly collected but the temperature will be continuously captured using the PicoLog Data Logger. The data produced by the PicoLog will be compared with the vibration spectra, in order to quantify the effectiveness of the two condition monitoring techniques. The results from this experiment will reveal more understanding on the dependent and independent roles of vibration and temperature in predicting and diagnosing machine faults.

The anticipated key issues will be pointed out in order to produce good condition monitoring practise for worm gearbox. A prediction of gear oil failure and lifespan are much depending on few characterizations. These are:

- The correlations of mechanical vibration and temperature will be much depending on operation speed.
- (ii) The contaminated gear oil will show an abnormal vibration trending compared to the normal gear oil.



1.6 Potential Contribution

So far, vibration-based condition monitoring of a worm gearboxes has been mostly studied on the wear pattern. An attention to run the gearboxes with contaminated gear oil need to be done to monitor the worm gear vibration behavior. Therefore, the first step in successfully implementing of condition monitoring of worm gear is to establish the baseline behavior of a healthy gear with a suggested gear oil by most industrial practise.

PERPUSTAKAAN TUNKU TUN AMINAH PERPUSTAKAAN

CHAPTER 2

LITERATURE REVIEW AND THEORY

2.1 Introduction



Diagnosis and fault detection of gear transmission systems have attracted wide attention in order to reduce breakdown in the industrial rotating machinery. Some researches had been done as to develop the understanding and correlations of contaminated gear oil which generally will contributed to gears failure. Introducing a proper condition monitoring techniques will surely extend the life span of the rotating equipment.

Most of the industrial practice are much rely on the vibration spectral data analysis as a primary tools for assessing machine condition beside to some others predictive tools which available in the market (Yu, 2011). In order to produce proper report of individual machine, a scientific approach need to be considered. All the data gathered by the analyst will be examined along with their deviation comparing to the appropriate baseline values (Watts, 2011). At the end, both parties which are end users and analyst need to understand that the collected data are all facts but the resulting diagnosis and or repair recommendation may be speculation. Some of the predictive tools may need qualified personnel to interpret the data correctly.

2.2 Condition Monitoring Techniques

The condition monitoring techniques philosophy consists of scheduling maintenance activities only when a functional failure is detected. Mechanical and operational conditions are periodically monitored, and when unhealthy trends are detected, the troublesome parts in the machine are identified and scheduled for maintenance. The machine would then be shut down at a time when it is most convenient, and the damaged components would be replaced. If left unattended, these failures could result in costly secondary failures. One of the advantages of this approach is that the maintenance events can be scheduled in an orderly fashion. It allows for some lead-time to purchase parts for the necessary repair work and thus reducing the need for a large inventory of spares. Since maintenance work is only performed when needed, there is also a possible increase in production capacity. A possible disadvantage is that maintenance work may actually increase due to an incorrect assessment of the deterioration of machines. According to (Serrato et al., 2007), 50-80% of bearing components faults are related to deficient lubrication, resulting from inadequate lubricant use, lack or excess of lubricant. To track the unhealthy trends in vibration, temperature or lubrication requires the facility to acquire specialized equipment to monitor these parameters and provide training to personnel (or hire skilled personnel).



Various condition monitoring methods can be used for the diagnosis gearbox and gear failures. The methods are namely classified as vibration analysis, oil sampling analysis, thermograph monitoring, acoustic monitoring and electric motor current monitoring. Figure 1 below shows the summary of types and causes of gear failures. (MTII Troubleshooting Gear Drives, 2008)

SUMMARY OF TYPES & CAUSES OF GEAR FAILURES

Types of Failure, %

Breakage, total 32.8 Fatigue breakage, teeth 32.8 Fatigue breakage, bore 4.0 Overload breakage, teeth 19.5 Overload breakage, bore 0.6 Chipping, teeth 4.3	61.2
Surface fatigue, total Pitting Spalling Pitting and spalling	20.3
Wear, total	13.2
Plastic flow, total	5.3

Causes of Failure, %

Service-related causes, total 11.2 Improper assembly 16.0 Continual overloading 25.0 Impact loading 8.9 Bearing failure 10.7 Foreign material 1.4 Operator error 0.3 Abusive handling 1.2		75% can be influenced by customer
Heat treatment, total 0.5 Excessive core hardness 0.5 Insufficient core hardness 2.0 Excessive case depth 1.8 Insufficient case depth 4.8 Improper hardening 5.9 Improper tempering 1.0 Distortion 0.2	16.2	
Design-related causes, total 2.8 Improper design 2.8 Improper material selection 1.6 Specification of unsuitable heat treatment 2.5 Manufacturing-related causes, total 0.7	6.9	25% cannot be influenced by customer
Tool marks or notches 0.7 Material-related causes, total 0.1 Forging defects 0.1 Steel defects 0.5 Mixed steel, wrong composition 0.2	0.8	

Figure 1: Summary of types and causes of gear failures, (MTII Troubleshooting Gear

Drives, 2008).

2.2.1 Vibration Analysis

Vibration analysis is the most widely used to predict rotating machinery failure in the industrial application. Vibration analysis is often done based on the bearing vibration signal measured by an accelerometer, usually piezoelectric sensor which convert the vibration signal to electrical signal. Since the abnormal vibration of rotary machines is the first sensory effect of rotary component failure, vibration analysis is widely employed in the industry. The fault vibration signal generated by the interaction between a damaged area and a rolling surface occurs regardless of the defect type. Consequently, a vibration analysis can be employed for the diagnosis of all types of faults, either localized or distributed. Furthermore, low-cost sensors, accurate results, simple setups, specific AN TUNKU TUN AMANA information on the damage location, and comparable rates of damage are other benefits of the vibration measurement method.

2.2.2 Oil Sampling Analysis



The interpretation of oil sampling analysis method is based on the presence of metallic or wear particles in the sampling lubricant collected (Wang, 2008). One of the most common and simplest methods to apply the oil debris monitoring techniques is the magnetic sump plug. The magnetic plug will captured the metallic debris from the lubricant oil path flow. The amount of metallic debris captured will represent the wear trend of the machine provided it has the baseline information. The limitations of wear debris magnetic sump plug method are only on the metallic debris and not extend to the other wear component such as seals and glands. To overcome the situation the oil sampling analysis can be conducted, where the spectrographic analysis of the different metallic elements and other wear components in the lubricant could be extracted and would able to facilitate the location of the fault. The another enhanced method of wear debris monitoring, uses magnets of increasing strength to separate wear particles from the

lubricating oil and grades particles by size and magnetic susceptibility on glass slide. However the interpretation of the results is subjective and the ferrographic method cannot be applied for on-line application, (Whittington et al., 1992) and (Roylance, 2005).

2.2.3 Thermograph Monitoring

Another method of condition monitoring is thermograph. Any defects occurred at bearing or gears in the rotating equipment will generate excessive heat that would be easily detected by the thermograph devices. Monitoring the temperature of a bearing housing or lubricant is the simplest method for fault detection in rotary machines.

Taking an example of mineral base gear oil which will begin to oxidize at temperatures that consistently run at 71°C, (MTII Troubleshooting Gear Drives 2008). A few parameters that will contribute to increase the gear oil temperature are excessive sludge, low lubricant levels, oil contamination, bearing failure or damaged gear teeth. Monitoring temperature differences at multiple gearbox, thermograph devices will display various heat image contour. The most problematic gearbox will have the indication of high temperature which mainly due to lubricant, bearing and gears.



2.2.4 Acoustic Monitoring

The most effective acoustic-based bearing health monitoring is acoustic emission. It is a transient impulse generated by the rapid release of strain energy in solid material under mechanical or thermal stress (Vladimir, 1999). The detection of cracks is the prime application of acoustic emission; therefore, this technique can be used as a tool for condition monitoring of bearing faults and shaft cracks. The measurement of a machine's

sound can also be employed for detecting defects in bearings. Typically, the accuracy of these methods depends on sound pressure and sound intensity data.

2.2.5 Electric Motor Current Monitoring

The operating conditions of a machine can be monitored by analyzing the spectrum of the motor current. The changes in the electric background noise are associated with the changes in the mechanical components of the machine; therefore, fault signatures can be detected by motor current signal processing techniques.

2.3 Industrial Gearboxes



There are variety of gears composition and arrangement as to meet the industrial needs. Engineer has to put their kind attention to select the best gears arrangement either to serve load, speed or direction. Figure 2 shows type of gears and some of the gears composition are discussed below:

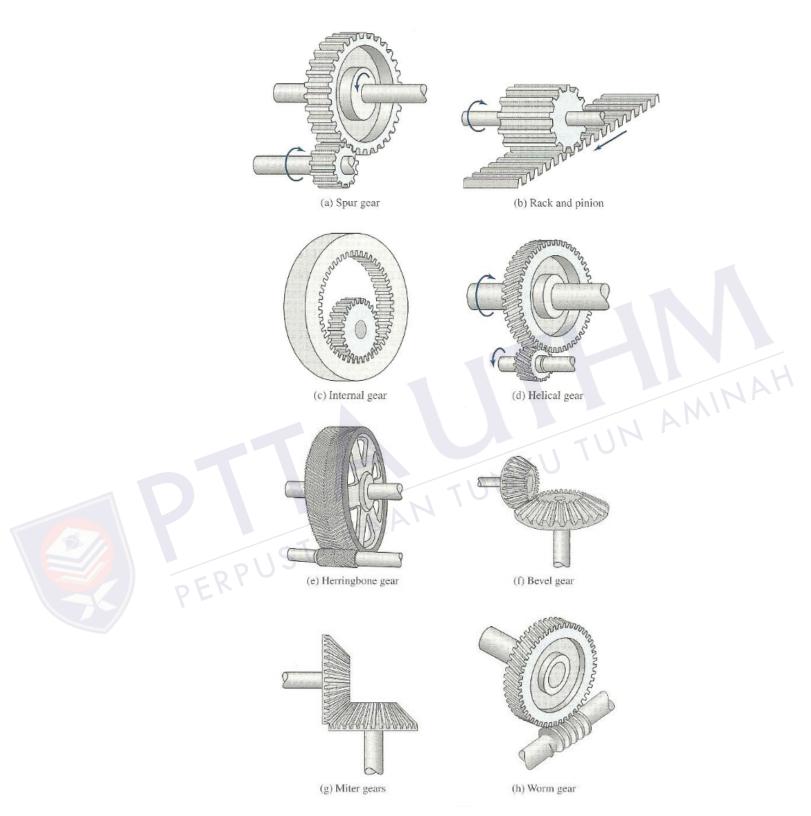


Figure 2: Type of gear, (Myszka, 2012).

2.3.1 Worm gear

A worm gear as shown in figure 3, consists of a spirally grooved screw moving against a toothed wheel. In this gear type, where the load is transmitted across sliding, rather than rolling surfaces, compounded or EP oils are necessary to provide effective lubrication.

Worm gears operate under difficult conditions, presenting unique lubrication demands. Worm gears are used in various industries and machinery applications. They are unique in their ability to achieve large speed reductions in a compact space. They can transmit high loads at high-speed ratios. Ratios of 20:1 up to 60:1 and higher are normally achieved.

There are three major types of worm gears:

- (i) Non-throated a helical gear with a straight worm. Tooth contact is a single moving point on the worm drive. This leads to high unit loads and wear
- (ii) Single-throated has concave helical teeth which wrap around the worm.This leads to line contact, permitting higher loads without excessive wear.
- (iii) Double-throated called a cone or hourglass. It has concave teeth both on the worm and helical gear. This increases from line contact area permitting increased loading and lower wear.



Figure 3: A screw driving a phosphor-bronze worm wheel gear.

Worm gears are commonly use in transmission design where its needs a compact, high reduction and relatively low speed output (Sharif, 2006). Prediction of the wear pattern in worm gears. The normal combination of hard/soft material is necessary as to avoid scuffing since a high degree of sliding between tooth wheel and worm gears are present. The usual configuration is hard steel for worm and a softer material like bronze for wheel, but then the wear rate of bronze gear teeth is much higher than that which might be expected in conventional gearing systems. Anyway, the worn off bronze gear can be replaced with a new one using the process of "*bedding in*".

2.3.2 Planetary Gear Transmission System



Planetary gearboxes are typically used in applications requiring a large reduction in speed at high loads, such as the final reduction in the main rotor gearbox of a industrial huge stirrer tank. A typical planetary reduction gearbox has three or more planet gears each meshing with a sun and a ring gear. The planetary gears are mounted onto the planetary carrier and contained within an internal toothed ring gear. Drive is provided via the sun gear, the ring gear is stationary and the axes of the planet gears are connected to a carrier which rotates in relation to both the sun gear and ring gear. The planet carrier provides the output of the planetary gear train. A single stage planetary gearset normally will consists of a sun gear, a ring gear, several planets and a carrier.

2.3.3 Parallel Gear Transmission System

The parallel gear system consists of a pinion gear and a driven gear. The pinion gear has a smaller number of teeth than the driven gear. Normally, a gear transmission system is designed to reduce the angular velocity in order to increase the output torque. In such a speed reduction gear transmission system, the pinion is connected with an input shaft, and the driven gear is connected with an output shaft.

2.4 Lubricant

As a general rules, oil temperature of enclosed gear drives should never exceed 89 °C – 93 °C and the best service will be obtained with at temperatures in a range of 50°C -60°C (MTII Troubleshooting Gear Drives, 2008).

2.4.1 Lubricant Viscosity Classification

AMINAI Through the years, lubricant users have been treated to a number of ways to designate viscosity grades of the lubricants used in manufacturing. They are:

- SAE (Society of Automotive Engineers) grades for gear oils and crankcases (i) (engines)
- (ii) AGMA (American Gear Manufacturers Association) grades for gear oils
- (iii) SUS (Saybolt Universal Seconds)
- cSt (kinematic viscosity in centistokes), and (iv)
- (v) absolute viscosity
- (vi) ISO VG (International Standards Organization Viscosity Grade)

Viscosity is the measure of the oil's resistance to flow (shear stress) under certain conditions. To simplify, the oil's viscosity represents the measure for which the oil wants to stay put when pushed (sheared) by moving mechanical components. Some of the oil viscosity comparison is shown as in Figure 4.

There are two viewpoints of the resistance to flow that the machine designer is interested in. One is the measure of how the fluid behaves under pressure, such as a pressurized hydraulic line. This property is called absolute viscosity (also known as dynamic viscosity) and is measured in centipoises (cP). The other consideration is how the fluid behaves only under the force of gravity which is called as centistokes. The two are related through the specific gravity of the fluid. To determine the centipoise of a fluid it is necessary to multiply the viscosity of the fluid times the specific gravity of the fluid, or measure it directly using an absolute viscometer. For the practitioner of industrial lubrication, the centistoke is the measure that will occupy most of our attention.

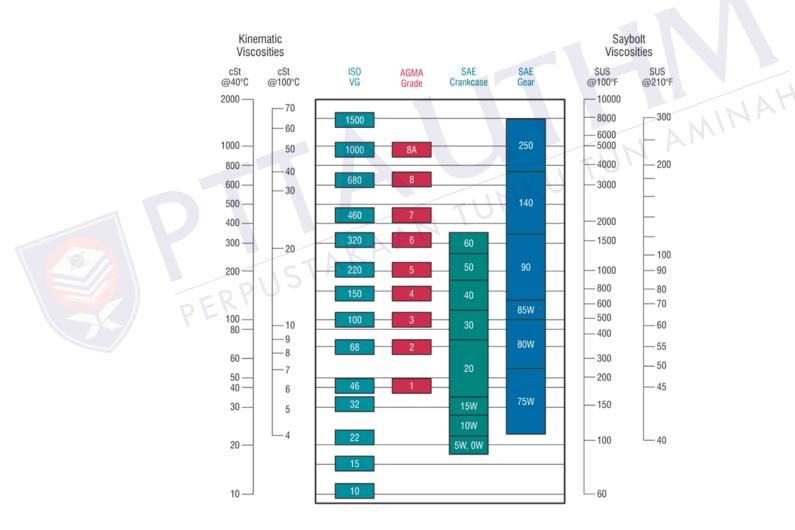


Figure 4: Comparative Viscosity Classification (http://www.machinerylubrication.com/Read/28681/gear-oil-life).

2.4.2 The Impact of lubricant on mechanical vibration

The thicker the oil thickness causes the level of vibration decrease. Author (Serrato, (2007), focus on three different viscosity grade (ISO 10, 32 and 68). Different ISO grade viscosity or temperature variation changes the vibration of the roller bearing. Previous study shows that oil temperature is proportional to the viscosity degree. High friction force (high viscosity) is related high loss of energy resulting in high heat generation. The low viscosity lubricants cause high vibration level.

Faults on roller bearing found because of 50-80% related to deficient lubrication, inadequate lubricant use, lack of lubricant, lubricant aging and presence of solid or liquid contaminant. The vibration energy of a bearing depends on surface irregularities, external loading, running speeds and lubricant viscosity. Lubricant viscosity causes reduction in vibration energy.



The vibration level is smaller as oil viscosity degree becomes higher, in contrast to the effect of temperature on oil viscosity. The testing time is major factor for gradual decrease in film thickness. Oil film thickness becomes smaller when the viscosity grade is changed to smaller values. Higher bearing vibration causes of smaller film thickness which related to lower oil viscosity.

According to author (Sharma *et al.*, 2008), the oil temperatures are maintained within the recommended range. When the particles come in contact with high-temperature zones these lead to formation of hard and abrasive particles. These on contact with the components cause generation of wear particles causing further reduced system performance. Therefore, to control wear and for increased performance, viscosity and contaminants are the contributing performance parameters.

REFERENCES

- Amarnath.M., Sujatha.C., Swarnamani.S., (2007). Experimental studies on the effect of reduction in gear tooth system, *Tribology international*, 42, pp.340-352.
- BLP, (2012),. Boetech Lubricate Product Data Sheet
- Brouwer, M.D., Gupta, L.A., Sadeghi, F., Peroulis, D., Adams, D., (2012). High temperature dynamics viscosity sensor for engine oil application, *Sensors and Actuators A:Physical*, 173,pp.102-107.
- Elforjani, M., Mba, D., Muhammad, A. Sire, A., (2012). Condition monitoring of worm gears.
- Ghafari, S.H., (2007).A Fault Diagnosis System for Rotary Machinery Supported by Rolling Element Bearings
- Goncalves, A.C., Cunha, R.C., Lago, D.F., (2006). *Maintance of a reducer by vibration* and wear particles analysis, Applications and case study, 12, pp118-132.

Infrared Training Center, (2010). Thermography Level 1 Course Manual

Myszka, D.H., (2012). Machines & Mechanisms Applied Kinematic Analysis.

- Kanaan, H.Y., Al-Haddad, K., Roy, G.,(2003). Analysis of the electrochemical vibrations in induction motor drives due to the imperfections of the mechanical transmission system, *Mathematics and Computers in Simulations*, 63, pp.421-433.
- Peng.Z., Kessissoglou.N.J., Cox.M., (2005). A study of the effect of contaminat particles in lubricants using wear debris and vibration condition monitoring technique, *Wear*,258,pp.1651-1662



- Roylance, B.J., Hunt, T.M., Wear Debris Analysis, Coxmoor Publishing Company, Oxford, UK, 1999.
- Serrato.R., Maru.M.M., Padovese.L.R.,(2007). Effect of lubricant viscosity grade on mechanical vibration of roller bearings, *Tribology International*, 40, pp.1270-1275.
- Sharif, K.J., Evans, H.P., Snidle, R.W., (2006). Prediction of the wear pattern in worm gears
- Sharma, B.C., Gandhi .O.P., (2008). Performance evaluation and analysis of lubricating oil using parameter profile approach, *Tribology International*, 60(3), pp.131-137.
- Su YT, Sheen YT, Lin MH, (1992). Signature analysis of roller bearing vibrationslubrication effect, Proceeding of the Institution of Mechanical Engineers Part C. J Mech Eng Sci 1992;206
- The Practical Handbook of Machinery Lubrication, 3rd Edition, L. Leugner, PP 63–69, 180–217.
- Vladimir, V., (1999). Detection and quantification of the gear tooth damage from the vibration and acoustic signatures
- Wang, R.Y., (2008) Bearing Fault Detection and Oil Debris Monitoring by Adaptive Noise Cancellation
- Watts, W., (2011). Machinery vibration analysis Facts v.s. Speculations
- Whittington, H.W., Flynn, B.W., and Mills, G.H., 1992, An online wear debris monitor. Measurement Science and Technology, vol.3.
- Yu, J., (2011). Early Fault Detection for Gear Shaft and Planetary Gear Basedon Wavelet and Hidden Markov Modeling

